



## Experimental Analysis of the Effect of Nanoparticles on Performance of the Air Conditioning Compressor

1. BANOTH SOMOJI\*, 2. Mr.B.Pavan naik\*\*, 3.Dr.M.Janardhan\*\*\*

\*M.Tech Student \*\*Associate Professor \*\*\*professor&Principal

Department of Mechanical Engg. Abdul Kalam Institute Of Technological Sciences,  
Kothagudem,Khammam (DT) -507120

### ABSTARCT:

It evaporating heat transfer is very important in refrigeration and air conditioning Systems. HFC 134a may be the alternative is often used in large scale Refrigerants in refrigeration equipment, such as domestic refrigerators And air conditioners. Despite the high temperatures of the global potential to Relatively high, HFC134a and stressed that it was a long time alternative refrigerants in many countries. By adding cooling nanoparticles to improve results in thermophysical properties and heat transfer characteristics of Cooling, thereby improving the cooling performance System. In these experiments the effect of using quotas using R134a system vapor pressure in the heat transfer coefficient of evaporation Research by heat transfer using CFD flow analysis Software. The experimental device constructed according National standards in India. Experimental studies suggest that nanorefrigerant system

normal operation of cooling. Hot Transport coefficients assessed using fluid flow heat Ranged 10-40 kW / m<sup>2</sup> And he was varied concentrations using nano CUO 0.05 to 1% and particle size of 10 to 70 nm. Results They suggest that increases the heat transfer coefficient with the evaporator Use nanoCuO.

**INTRODUCTION:** Rapid industrialization has led to unprecedented development and growth technological advances around the world. Global warming and today the ozone layer It has become a drain on the one hand and rising oil prices on the other hand, the main Challenges. Excessive use of fossil fuels leads to a decrease in its sharp, Nuclear power is not safe. In the face of an impending energy resources Crisis there is a need for the development of thermal systems that are efficient in energy use. Thermal systems such as refrigerators and air conditioners consume a lot of The electric energy. the development of energy



efficiency and air cooling is required air conditioning systems with cooling nature of use. The rapid progress of Nanotechnology can lead to the emergence of heat transfer fluids called new generation nanofluids. Nanofluids is willing to suspend nanometer-sized particles (1-100nm) in Conventional fluid and has a higher thermal conductivity than the base fluid. Nanofluids have the following characteristics compared with ordinary solid-liquid Comment. I) a greater heat transfer between particles and fluids due to the high Surface of the particles b) improve the stability and dispersion with prevailing Brownian Movement c) reduces particles clogged d) reduce the pump capacity compared to the base Fluid heat transfer equivalent. Based on the applications, the nanoparticles are Currently she carried out by a very wide range of materials, the most common of the new Jill be ceramic nanoparticles, which are the divided metal oxide Ceramics, such as oxides of titanium, zinc, aluminum and iron, for example, but not limited to prominent And nanoparticles silicate, generally in the form of nano clay flakes. By In addition to cooling of nanoparticles to improve results in thermophysical properties and characteristics of the refrigerant heat transfer, and therefore Improve the performance of the cooling system. The

vapor pressure Nanoparticles cooling system can be added to lubricants. HFC 134a may be an alternative cooling often used widely in refrigeration Equipment such as refrigerators and air conditioners. Despite the global Heat the potential for relatively high HFC134a, and stressed that the long-term alternative refrigerants in many countries. Experience vapor pressure It was carried out cooling pad test to calculate the cooling effect and laboratory apparatus performance testing to identify potential sites to promote cooling effect. Mathematical modeling to test section tube evaporator using partial differential equations. This has been done theoretical analysis Tests in the evaporator section. The use of a network program evaporator maneuver test section designed and use the program flow in the heat transfer analysis Make different concentrations of nanoparticles CUO.

## LITERATURE REVIEW:

technologies. Renew Sustain Energy Rev 2001;5(4): 343–72[1] they given initialisation of work that Absorption refrigeration was discovered by Nairn in 1777, though the first commercial refrigerator was only built and developed in 1823 by Ferdinand Carré, who also got several patents between 1859 and 1862 from introduction of a machine operating on ammonia–water. By 19th century, systems



operating on ammonia–water found wide application in residential and industrial refrigerators. Systems operating on lithium bromide–water were commercialized in the 1940's and 1950's as water chillers for large buildings air conditioning.

Horuz I. An alternative road transport refrigeration. *Tr. J. Of Engineering and Environmental Science* 1998;22:211-222.

[2]conducted experimental investigation into the effect on the performance of the IC engine of introducing the VAR system into the exhaust system and also the provision of appropriate off-road/slow running cooling systems, in order to take account of the reduction in exhaust gas flow in slow running traffic or stationary situations or when the vehicle is parked and cooling is still required. Built-in eutectic plates could provide temporary cooling under such conditions. Such plates could be recharged by redirecting the cooling effect from the main body to the eutectic plate during off-load periods of continuous full-load travel.

Alam [3] Shah A. A proposed model for utilizing exhaust heat to run automobile air-conditioner. The 2nd Joint International Conference on Sustainable Energy and Environment 2006. studied the possibility of operating a triple fluid vapour absorption system using engine exhaust power. From the analysis it was concluded that there is a possibility of operating a triple fluid system using engine exhaust power.

[4]S.U.S. Choi, ASME, 99(1995) showed that the addition of a small amount of nanoparticles (less than 1% by volume) to base fluid would increase the thermal conductivity of the fluid up to approximately two times. But thermal conductivity having the most important property is not easy to determine accurately by a single formula, but there are some experimental relations that could be used to estimate it. With increase in temperature, there is increase in thermal conductivity but there is abnormal behavior of the thermal conductivity at high temperatures which is related to the solubility of the nanoparticles. In the nanometrical size range, kinetics of dissolution of particles is enhanced due to the small size according to the Kelvin equation.

Eastman JA, Choi US, Thompson LJ, Lee S. “Enhanced thermal conductivity through the development of nanofluids. *Mater Res Soc Symp proc* 1996;457:3-11[5] concluded that report that thermal conductivity of ethylene glycol of nanofluids containing 0.3% volume fraction of copper particles can enhance thermal properties by 40% compared to that of EG (Ethylene Glycol) base fluid.



Z. Zhang and Q. Que, *Wear* 209, 8 (1997).[6] In automobile lubrication nanoparticles dispersed in mineral oils were reported to be effective in reducing wear & enhancing load carrying capacity [41]. Recently lots of researchers show their interest to enhance the tribological properties (such as load carrying capacity, wear resistance and friction reduction) of nanoparticle suspended lubricants. The vehicle life time as well as the performance will be increased by using the nanoparticle suspended lubricants. Osorio et al. investigated the tribological properties of CuO suspended lubricant.

[7] Wang X, Xu X, Choi SUS: Thermal conductivity of nanoparticle-fluid mixture. *J Thermophys Heat trans* 1999, 13:474-480 due to the low pressure operation compared with a 50/50 mixture of ethylene glycol and water, which is the universally used automotive coolant. The nanofluids has a high boiling point, and it can be used to increase the normal coolant operating temperature and then reject more heat through the existing coolant system and also contributed to a reduction in friction and wear. It is conceivable that greater improvement of savings could be obtained in the future but with time nanofluids degrade radiator material and Erosion of

radiator material will be there. Choi studied on the development of energy efficient nanofluids and smaller and lighter radiators. A major goal of the nanofluids project is to reduce the size and weight of the HV cooling systems by >10% thereby increasing fuel efficiency by >5%. Nanofluids enable the potential to allow higher temperature coolants and higher heat rejection in HVs. A higher temperature radiator could reduce the radiator size by perhaps 30%.

[8] Interagency working group on nano science, national nano technology initiative: Leading to the next industrial revolution, Technology National Science and Technology Council, USA, February (2000). Due to depletion of natural resources like natural gas, oil and water at faster rate and due to trends toward faster speeds it is need to improve the efficiency and the performance of automobile by using different methods. If this rate is continued, then we are at verge of extinction. Many methods like reducing the vehicle weight, improving the engine performance have been used and also under investigation. In the recent research it has been found that we can use the nanoparticles as fuel additives to improve the fuel economy as well as to reduce the exhaust emissions and also combustion stability. The scientists in nano

science and technology council in USA have achieved to increase 10-25% combustion efficiency by adding 0.5% of aluminum nanoparticles to a rocket's solid fuel.

[9] Eastman JA, Choi SUS, Li S, Yu W, Thompson LJ. "Anomalously increased effective thermal conductivities of ethylene glycol-based nanofluids containing copper nanoparticles". Appl Phys Lett 2001;78(6):718–20. Eastman et al found [24] that a "nanofluid" consisting of copper nanometer-sized particles dispersed in ethylene glycol has a much higher effective thermal conductivity than either pure ethylene glycol or ethylene glycol containing the same volume fraction of dispersed oxide nanoparticles. Thermal conductivity of ethylene glycol can be increased by 40 % for a nanofluids consisting of ethylene glycol containing approximately 0.3 volume % Cu nanoparticles of mean diameter <10 nm.

[10] Xie H, Wang J, Xi T, Liu Y, Ai F: Thermal conductivity enhancement of suspensions containing nanosized alumina particles. J Appl Phys 2002, 91:4568-4572 Xie et al. used 60.4-nm-sized particles, observed higher thermal conductivity enhancement for larger nanoparticles in ethylene glycol-based nanofluids. In the case

of Xie et al. , the researchers used 60.4-nm-sized Al<sub>2</sub>O<sub>3</sub> dispersed in water and prepared stable solution by adjusting pH. The nanoparticles are de-agglomerated by using an ultrasonic disrupter after mixing with a base fluid and were homogenized by using magnetic force agitation. The enhancement observed was 21% for 5% volume fraction and 14% at 3.2% volume fraction.

### METHODOLOGY:

Two-step method is the most widely used method for preparing Nano fluids. Nanoparticles used in this method are first produced as dry powders by chemical or physical methods. Then, the Nano sized powder will be dispersed into Mineral oil in the second processing step with the help of intensive magnetic force agitation and ultrasonic agitation. The schematics of magnetic stirrer and magnetic beads are shown in the figures:



Figure 3.1: Magnetic stirrer

Lubricating oil with Nano particles is placed in a beaker on the stirrer with a magnetic bead in it.



Figure: 3.2 Different Magnetic Beads

Two-step method is the most economic method to produce Nano fluids in large scale, because Nano powder synthesis techniques have already been scaled up to industrial production levels.

Due to the high surface area and surface activity, nanoparticles have the tendency to aggregate. The important technique to enhance the stability of nanoparticles in fluids is the use of surfactants. However, the functionality of the surfactants under high temperature is also a big concern, especially for high-temperature applications. Effects of Nano particles on air-conditioning cannot be isolated if surfactants are used.

Amount of lubricating oil used for the experiment is 650ml and the amount of  $\text{TiO}_2$  Nano

particles used is 0.2772 grams and the amount of  $\text{Al}_2\text{O}_3$  Nano particles used is 0.1126 grams.

#### **SPECTRAL ABSORBENCY ANALYSIS**

Spectral absorbency analysis is an efficient way to evaluate the stability of Nano fluids. In general, there is a linear relationship between the absorbency intensity and the concentration of nanoparticles in fluid. Experiments evaluated the dispersion characteristics of  $\text{TiO}_2$  and  $\text{Al}_2\text{O}_3$  suspension using the conventional sedimentation method with the help of absorbency analysis by using a spectrophotometer after the suspension deposited for 24 h and 7 days. If the Nano materials dispersed in fluids have characteristic absorption bands in the wavelength 190–1100 nm, it is an easy and reliable method to evaluate the stability of Nano fluids using UV spectral analysis. The variation of particle concentration of Nano fluids with sediment time can be obtained by the measurement of absorption of Nano fluids, because there is a linear relation between the nanoparticle concentration and the

absorbance of suspended particles. The outstanding advantage comparing to other methods is that UV spectral analysis can present the quantitative concentration of Nano fluids. It is believed that the stability of Nano fluids was strongly affected by the characteristics of the suspended particles and the base fluid such as particle morphology.

The intensity of transmitted radiation is measured using transducers. Intensity of incident radiation is known. Absorption is defined as the logarithm of ratio of intensities of incident and transmitted radiation.

$$\text{Absorption} = \log(I_0/I)$$

Where  $I_0$  is the intensity of incident radiation and  $I$  is the intensity of transmitted radiation

## **RESULTS AND DISCUSSIONS**

➤ Thermal conductivity of Nano particles mixed in lubricating oil is found to be greater than that of the base fluid. This result agrees with that of the results obtained in literature. This is to be expected as thermal conductivity of metals/

metal oxides nano particles is higher than the base low thermal conductivity mineral oil.

- Analysis of dispersion characteristics of nanoparticles in lubricating oil using spectrophotometer shows that the Nano fluid is not stable and nano particles form sediments on the 1<sup>st</sup> day and 7<sup>th</sup> day, this is perhaps the reason we got only slight improvement in EER as nanoparticles had settled in the crank casing of the compressor. In order to overcome the problem of sedimentation of nano particles, surfactants can be used in lubricating oil.
- Compressor performance tests indicate that increase in EER is 0.1% when  $\text{TiO}_2$  Nano particles are mixed with Mineral oil and 1.5% decrease in EER, when  $\text{Al}_2\text{O}_3$  Nano particles are mixed with Mineral oil which is not encouraging. So, further experimentation is required with higher concentrations of nanoparticles with smaller sizes of Nano particles; surfactants can also be added to the Nano Fluids and EER checked.

**For Base Oil: Mineral oil**



Tecomseh Products India Pvt.Ltd,Hyderabad

Calorimeter Test Results

Cal-2 Q.F.7.3-01-15

| TEST DOCUMENTATION  |              |                     |                |
|---------------------|--------------|---------------------|----------------|
| TEST PURPOSE        | DEV SAMPLE   | REPORT NO.          | C12081016 - L2 |
| TEST DATE           | 8/11/2012    | PROJECT NO.         |                |
| BILL OF MATERIAL    | BS12081016   | MODEL NO.           | RNB5528BNA     |
| COOLING METHOD      | 425 CFM      | DISPLACEMENT CC/REV | 28             |
| SERIAL NUMBER       | 146417       | MOTOR MANUFACTURER  | TPIPL HYD      |
| CODE DATE           |              | MOTOR SPECS.        | PSC            |
| REFRIGERANT         | R410A        | PHASE               | Single Phase   |
| REQUESTING ENGINEER | M.P.REDDY    | FREQUENCY Hz        | 50             |
| TESTED BY           | MD.GHOUSE    | CUSTOMER NOTFN. NO. |                |
| TEST NAME           | ASHRAE-R410a | TEST NUMBER         | 5547           |

| TEST MEASUREMENTS  |              |        |           |        | DEV SAMPL TEST WITH (MINERAL OIL) |
|--------------------|--------------|--------|-----------|--------|-----------------------------------|
| PRESSURES          | ACTUAL VALUE |        | SET VALUE |        | MINERAL OIL + TITANIUM DI -OXIDE. |
|                    | BAR (A)      | PSIA   | BAR (A)   | PSIA   |                                   |
| SUCTION PRESSURE   | 9.9336       | 144.07 | 9.954     | 144.37 | O.C.R - 1.47%                     |
| DISCHARGE PRESSURE | 33.846       | 490.90 | 33.85     | 490.95 |                                   |
| CAL. OUT PRESSURE  | 9.908        | 143.70 |           |        |                                   |

| TEMPERATURES          |      |       |      |       |
|-----------------------|------|-------|------|-------|
|                       | °C   | °F    | °C   | °F    |
| EVAPORATING TEMP.     | 7.1  | 44.8  | 7.2  | 45.0  |
| CONDENSING TEMP.      | 54.5 | 130.1 | 54.5 | 130.1 |
| DEGREE OF SUPERHEAT   | 27.9 | 50.3  | 27.8 | 50.0  |
| DEGREE OF SUBCOOLING  | 8.4  | 15.1  | 8.4  | 15.2  |
| RETURN GAS TEMP.      | 36.0 | 95.0  | 35.0 | 95.0  |
| LIQUID TO EXPN. VALVE | 46.1 | 115.0 | 46.1 | 115.0 |
| CAL. OULET TEMP.      | 35.1 | 95.1  | 35.0 | 95.0  |
| COMPR. CHAMBER AMB.   | 34.9 | 94.9  | 35.0 | 95.0  |
| Top Shell             | 73.9 | 165.1 |      |       |
| Bottom Shell          | 58.1 | 136.6 |      |       |
| Middle at shell       | 70.5 | 159.0 |      |       |
| Discharge Line        | 87.1 | 188.7 |      |       |

| ELECTRICAL MEASUREMENTS |              |           |
|-------------------------|--------------|-----------|
| PARAMETER               | ACTUAL VALUE | SET VALUE |
| FREQUENCY               | 49.99 Hz     | 50 Hz     |
| COMPRESSOR VOLTAGE1     | 219.44 Volts | 220 Volts |
| COMPR. CURRENT1         | 10.79 Amp.   |           |
| COMPR. POWER            | 2304 Watts   |           |
| COMPR. PF. MEASURED     | 0.97         |           |
| CAL. HEATER ENERGY      | 6683 Watts   |           |
| RUN CAPACITOR USED      | 45MFD        |           |
| ECR                     | 316.32 Volts |           |
| ECH                     | 245.38 Volts |           |
| START WDG. CURRENT      | 4.68 Amp.    |           |
| MAIN WINDING CURRENT    | 8.5 Amp.     |           |
| MOTOR SPEED             | 2882 RPM     |           |

| REFRIGERATION RESULTS |               |                |       |
|-----------------------|---------------|----------------|-------|
| MASS FLOW RATE, CALC. | 2.291 Kg/min  | 303.06 Lbs/hr  |       |
| MASS FLOW RATE, MEAS. | 2.276 Kg/min  | 301.14 Lbs/hr  |       |
| MASSFLOW AGREEMENT    | 0.63 %        |                |       |
| CORRECTED CAPACITY    | 6745.85 Watts | 23037.08 Btu/h |       |
| CORRECTED COMP. POWER | 2304.46 Watts |                |       |
| VOLUMETRIC EFFICIENCY | 88.30 %       |                |       |
| ISENTROPIC EFFICIENCY | 65.89 %       | EER            | 10.00 |

| DEVIATION ANALYSIS   |         |          |           |
|----------------------|---------|----------|-----------|
|                      | NOMINAL | ACTUAL   | DEVIATION |
| COMPR. POWER, Watts  | 2320    | 2304.46  | -0.7      |
| COMPR. CURRENT, Amp. | 11.5    | 10.79    | -6.2      |
| CAPACITY, Btu/Hr     | 23400   | 23037.08 | -1.6      |
| EER (Btu/Wh)         | 10.10   | 10.00    | -1.0      |

Winding Temperature 105.40°C Checked By

**For Nano Fluid: Mineral oil + 0.01(v/v)% TiO<sub>2</sub>:**



Tecomseh Products India Pvt.Ltd,Hyderabad

Calorimeter Test Results

Cal-2 Q.F.7.3-01-15

| TEST DOCUMENTATION  |              |                     |               |
|---------------------|--------------|---------------------|---------------|
| TEST PURPOSE        | DEV SAMPLE   | REPORT NO.          | C12081016- L3 |
| TEST DATE           | 9/11/2012    | PROJECT NO.         |               |
| BILL OF MATERIAL    | BS12081016   | MODEL NO.           | RNB5528BXC    |
| COOLING METHOD      | 425 CFM      | DISPLACEMENT CC/REV | 28            |
| SERIAL NUMBER       | 146417       | MOTOR MANUFACTURER  | TPIPL HYD     |
| CODE DATE           |              | MOTOR SPECS.        | PSC           |
| REFRIGERANT         | R410A        | PHASE               | Single Phase  |
| REQUESTING ENGINEER | M.P.REDDY    | FREQUENCY Hz        | 50            |
| TESTED BY           | MD.GHOUSE    | CUSTOMER NOTFN. NO. |               |
| TEST NAME           | ASHRAE-R410a | TEST NUMBER         | 5548          |

| TEST MEASUREMENTS  |              |        |           |        | DEV SAMPLE (TEST WITH)            |
|--------------------|--------------|--------|-----------|--------|-----------------------------------|
| PRESSURES          | ACTUAL VALUE |        | SET VALUE |        | MINERAL OIL + TITANIUM DI -OXIDE. |
|                    | BAR (A)      | PSIA   | BAR (A)   | PSIA   |                                   |
| SUCTION PRESSURE   | 9.9382       | 144.14 | 9.954     | 144.37 | O.C.R - 2.4%                      |
| DISCHARGE PRESSURE | 33.862       | 491.13 | 33.86     | 491.10 |                                   |
| CAL. OUT PRESSURE  | 9.912        | 143.76 |           |        |                                   |

| TEMPERATURES          |      |       |      |       |
|-----------------------|------|-------|------|-------|
|                       | °C   | °F    | °C   | °F    |
| EVAPORATING TEMP.     | 7.1  | 44.9  | 7.2  | 45.0  |
| CONDENSING TEMP.      | 54.5 | 130.2 | 54.5 | 130.2 |
| DEGREE OF SUPERHEAT   | 27.9 | 50.2  | 27.8 | 50.0  |
| DEGREE OF SUBCOOLING  | 8.4  | 15.2  | 8.4  | 15.2  |
| RETURN GAS TEMP.      | 35.0 | 95.0  | 35.0 | 95.0  |
| LIQUID TO EXPN. VALVE | 46.1 | 115.0 | 46.1 | 115.0 |
| CAL. OULET TEMP.      | 35.0 | 95.0  | 35.0 | 95.0  |
| COMPR. CHAMBER AMB.   | 35.0 | 95.0  | 35.0 | 95.0  |
| Top Shell             | 71.2 | 160.2 |      |       |
| Bottom Shell          | 54.5 | 130.2 |      |       |
| Middle at shell       | 78.9 | 174.0 |      |       |
| Discharge Line        | 87.6 | 189.6 |      |       |

| ELECTRICAL MEASUREMENTS |              |           |
|-------------------------|--------------|-----------|
| PARAMETER               | ACTUAL VALUE | SET VALUE |
| FREQUENCY               | 49.996 Hz    | 50 Hz     |
| COMPRESSOR VOLTAGE1     | 220.26 Volts | 220 Volts |
| COMPR. CURRENT1         | 10.772 Amp.  |           |
| COMPR. POWER            | 2304 Watts   |           |
| COMPR. PF. MEASURED     | 0.97         |           |
| CAL. HEATER ENERGY      | 6676 Watts   |           |
| RUN CAPACITOR USED      | 45MFD        |           |
| ECR                     | 316.62 Volts |           |
| ECH                     | 245.26 Volts |           |
| START WDG. CURRENT      | 4.7 Amp.     |           |
| MAIN WINDING CURRENT    | 8.52 Amp.    |           |
| MOTOR SPEED             | 2880 RPM     |           |

| REFRIGERATION RESULTS |               |                |           |
|-----------------------|---------------|----------------|-----------|
| MASS FLOW RATE, CALC. | 2.296 Kg/min  | 303.76 Lbs/hr  |           |
| MASS FLOW RATE, MEAS. | 2.300 Kg/min  | 304.34 Lbs/hr  |           |
| MASSFLOW AGREEMENT    | -0.19 %       |                |           |
| CORRECTED CAPACITY    | 6756.29 Watts | 23072.72 Btu/h |           |
| CORRECTED COMP. POWER | 2304.18 Watts |                |           |
| VOLUMETRIC EFFICIENCY | 88.49 %       |                |           |
| ISENTROPIC EFFICIENCY | 66.04 %       | EER            | 10.01     |
| DEVIATION ANALYSIS    |               |                |           |
|                       | NOMINAL       | ACTUAL         | DEVIATION |
| COMPR. POWER, Watts   | 2320          | 2304.18        | -0.7      |
| COMPR. CURRENT, Amp.  | 11.5          | 10.772         | -6.3      |
| CAPACITY, Btu/Hr      | 23400         | 23072.72       | -1.4      |
| EER (Btu/Wh)          | 10.10         | 10.01          | -0.9      |

Winding Temperature 102.76°C Checked By

**For Nano Fluid: Mineral oil + 0.01(v/v)% Al<sub>2</sub>O<sub>3</sub>:**





Tecumseh Products India Pvt.Ltd,Hyderabad

Calorimeter Test Results

Cal-2 Q.F.7.3-01-15

| TEST DOCUMENTATION  |              |                     |              |
|---------------------|--------------|---------------------|--------------|
| TEST PURPOSE        | Dev sample   | REPORT NO.          | C/2081016-L4 |
| TEST DATE           | 9/11/2012    | PROJECT NO.         |              |
| BILL OF MATERIAL    | BS/2081016   | MODEL NO.           | RNASS28BXC   |
| COOLING METHOD      | 425 CFM      | DISPLACEMENT COEFF  | 28           |
| SERIAL NUMBER       | 146417       | MOTOR MANUFACTURER  | TIPLI, HYD   |
| CODE DATE           |              | MOTOR SPECS.        | PSC          |
| REFRIGERANT         | R410A        | PHASE               | Single Phase |
| REQUESTING ENGINEER | M.P. REDDY   | FREQUENCY Hz        | 50           |
| TESTED BY           | R.CHANDRA    | CUSTOMER NOTFN. NO. |              |
| TEST NAME           | ASHRAE-R410a | TEST NUMBER         | 5549         |

| TEST MEASUREMENTS  |              |        |           |        |
|--------------------|--------------|--------|-----------|--------|
| PRESSURES          | ACTUAL VALUE |        | SET VALUE |        |
|                    | BAR (A)      | PSIA   | BAR (A)   | PSIA   |
| SUCTION PRESSURE   | 9.9364       | 144.12 | 9.3541    | 144.37 |
| DISCHARGE PRESSURE | 33.854       | 491.01 | 33.86     | 491.10 |
| CAL. OUT PRESSURE  | 9.912        | 143.76 |           |        |

Tested with Al2O3 oil  
Mineral oil- Aluminium oxide.  
O.C.R. - 1.7%

| TEMPERATURES          |      |       |      |       |
|-----------------------|------|-------|------|-------|
|                       | °C   |       | °F   |       |
|                       | IC   | IF    | IC   | IF    |
| EVAPORATING TEMP      | 7.1  | 44.8  | 7.2  | 45.0  |
| CONDENSING TEMP       | 54.5 | 130.1 | 54.5 | 130.2 |
| DEGREE OF SUPERHEAT   | 27.8 | 82.0  | 27.8 | 82.0  |
| DEGREE OF SUBCOOLING  | 8.5  | 15.2  | 8.4  | 15.2  |
| RETURN GAS TEMP.      | 35.0 | 95.0  | 35.0 | 95.0  |
| LIQUID TO EXPN. VALVE | 46.1 | 114.9 | 46.1 | 115.0 |
| CAL. OULET TEMP.      | 35.0 | 95.0  | 35.0 | 95.0  |
| COMPR. CHAMBER AMB.   | 34.9 | 94.9  | 35.0 | 95.0  |
| Top Shell             | 71.3 | 160.3 |      |       |
| Bottom Shell          | 58.8 | 137.8 |      |       |
| Middle at shell       | 75.0 | 167.0 |      |       |
| Discharge Line        | 86.8 | 188.3 |      |       |

| ELECTRICAL MEASUREMENTS |              |           |
|-------------------------|--------------|-----------|
| PARAMETER               | ACTUAL VALUE | SET VALUE |
| FREQUENCY               | 50.026 Hz    | 50 Hz     |
| COMPRESSOR VOLTAGE1     | 215.5 Volts  | 220 Volts |
| COMPR. CURRENT1         | 10.916 Amp.  |           |
| COMPR. POWER            | 2330 Watts   |           |
| COMPR. PF MEASURED      | 0.97         |           |
| CAL. HEATER ENERGY      | 6642 Watts   |           |
| RUN CAPACITOR USED      | 453µF        |           |
| ECR                     | 316.28 Volts |           |
| ECH                     | 244.4 Volts  |           |
| START WDG. CURRENT      | 4.7 Amp.     |           |
| MAIN WINDING CURRENT    | 8.56 Amp.    |           |
| MOTOR SPEED             | 2860 RPM     |           |

| REFRIGERATION RESULTS |               |                |
|-----------------------|---------------|----------------|
| MASS FLOW RATE, CALC. | 2.284 Kg/min  | 302.18 Lbs/hr  |
| MASS FLOW RATE, MEAS. | 2.297 Kg/min  | 303.92 Lbs/hr  |
| MASSFLOW AGREEMENT    | -0.58%        |                |
| CORRECTED CAPACITY    | 6721.90 Watts | 22955.28 Btu/h |
| CORRECTED COMP. POWER | 2329.72 Watts |                |
| VOLUMETRIC EFFICIENCY | 88.69%        |                |
| ISENTROPIC EFFICIENCY | 84.98%        | EER            |
|                       |               | 9.85           |

| DEVIATION ANALYSIS   |         |         |           |
|----------------------|---------|---------|-----------|
|                      | NOMINAL | ACTUAL  | DEVIATION |
| COMPR. POWER, Watts  | 2320    | 2329.7  | 0.4       |
| COMPR. CURRENT, Amp. | 11.5    | 10.91   | -5.1      |
| CAPACITY, Btu/Hr     | 23400   | 22955.3 | -1.9      |
| EER (Btu/Wh)         | 10.09   | 9.85    | -2.3      |

WINDING TEMPERATURE 106.7°C

Checked By

## CONCLUSIONS

- Spectroscopic analysis of nanoparticles added to lubricant oil shows that sediments start forming on the 7<sup>th</sup> day indicating that Nano particles are not fairly well dispersed in the base fluid. Surfactants may be added to enhance the dispersal level.
- Thermal conductivity of Nano fluids (TiO<sub>2</sub>, Al<sub>2</sub>O<sub>3</sub> Nano particles added to Mineral oil) is greater than that of the base fluid. This is to be expected as thermal conductivity of metals/metal

oxides nano particles added is higher than that of the base mineral oil. This is consistent with literature which reports an increase in Thermal Conductivity when Nano particles are added to water/Ethylene Glycol.

- The reproducibility of the air conditioning test facility was checked by repeating one case twice. Results showed agreement of EER values within ± 0.1%. The Energy Efficiency Ratio for TiO<sub>2</sub> spiked lubricant increased by 0.1% and for Al<sub>2</sub>O<sub>3</sub>, spiked lubricant, EER decreased by 1.5%. It is concluded that significant results were not obtained. It is inferred that in order to get an increase in EER, the nano particles concentration has to be increased and size of nano particles has to be decreased and tests have to be conducted afresh.

Also, care should be taken that lubricating oil left over in the crank case of the compressor housing after a particular experiment should be removed totally and filled afresh for next experiment, to avoid contamination

**REFERENCES:** 1. Adi T. Utomo, Heiko



Poth, Phillip T. Robbins, Andrzej W. Pacek “[Experimental and theoretical studies of thermal conductivity, viscosity and heat transfer coefficient of titania and alumina nanofluids](#)” International Journal of Heat and Mass Transfer, Volume 55, Issues 25–26, December 2012, Pages 7772-7781

2. Cherng-Yuan Lin, Jung-Chang Wang, Teng-Chieh Chen “[Analysis of suspension and heat transfer characteristics of Al<sub>2</sub>O<sub>3</sub> nanofluids prepared through ultrasonic vibration](#)” Applied Energy, Volume 88, Issue 12, December 2011, Pages 4527-4533

3. I.M. Mahbulul, S.A. Fadhilah, R. Saidur, K.Y. Leong, M.A. Amalin “Thermophysical properties and heat transfer performance of Al<sub>2</sub>O<sub>3</sub>/R-134a nanorefrigerants” International Journal of Heat and Mass Transfer, Volume 57, Issue 1, 15 January 2013, Pages 100-108

4. R. Krishna Sabareesh, N. Gobinath, V. Sajith, Sumitesh Das, C.B. Sobhan “[Application of TiO<sub>2</sub> nanoparticles as a lubricant-additive for vapor compression refrigeration systems – An experimental investigation](#)” International Journal of Refrigeration, Volume 35, Issue 7, November 2012, Pages 1989-1996

5. R. Saidur , S.N. Kazi , M.S. Hossain ,

M.M. Rahman , H.A. Mohammed “A review on the performance of nanoparticles suspended with refrigerants and lubricating oils in refrigeration systems” Volume 15, Issue 1, January 2011, Pages 310-323

6. Ruixiang Wang, Qingping Wu, Yezheng Wu “[Use of nanoparticles to make mineral oil lubricants feasible for use in a residential air conditioner employing hydro-fluorocarbons refrigerants](#)” Energy and Buildings, Volume 42, Issue 11, November 2010, Pages 2111-2117

7. Sheng-shan Bi, Lin Shi \*, Li-li Zhang, Application of nanoparticles in domestic refrigerators Volume 28, Issues 14–15, October 2008, Pages 1834-1843

8. Shengshan Bi, Kai Guo, Zhigang Liu, Jiangtao Wu “[Performance of a domestic refrigerator using TiO<sub>2</sub>-R600a nano-refrigerant as working fluid](#)”, Energy Conversion and Management”, Volume 52, Issue 1, January 2011, Pages 733-737

9. Xiang-Qi Wang, Arun S. Mujumdar, Heat transfer characteristics of nanofluids: a review International Journal of Thermal Sciences, Volume 46, Issue 1, January 2007, Pages 1-19